

(Following Paper ID and Roll No. to be filled in your Answer Book)

PAPER ID : 4032

Roll No.

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B.Tech.

(SEM. I) ODD SEMESTER THEORY EXAMINATION 2012-13 MECHANICAL ENGINEERING

Time : 3 Hours

Total Marks : 100

Note :—(1) Attempt all questions. Assume any missing data.
(2) Steam tables and charts are allowed.

- Attempt any **FOUR** parts of the following : (4×5=20)
 - Define the following terms :—
 - Thermodynamic Equilibrium.
 - Quasi static Process.
 - A fluid system undergoes a non flow frictionless process following the pressure volume relation as $p = \frac{5}{V} + 1.5$ where p is in bar and V is in m^3 . During the process the volume changes from $0.15 m^3$ to $0.05 m^3$ and system rejects 45 kJ of heat. Determine :
 - Change in internal energy.
 - Change in enthalpy.
 - A domestic food refrigerator maintains a temperature of $-12^\circ C$. The ambient air temperature is $35^\circ C$. If heat leaks into the freezer at the continuous rate of 2 kJ/S determine the least power necessary to pump this heat out continuously.
 - A heat pump operates between two identical bodies which are at temperature T_1 and cools one of the bodies to a temperature T_2 ($T_2 < T_1$). Prove that for this operation

the minimum work required by the heat pump is given by

$$W = C_p \left(\frac{T_1^2}{T_2} + T_2 - 2T_1 \right)$$

where C_p is the specific heat which is same for both the bodies.

(e) Explain the following terms relating to steam formation :

- (i) Sensible heat of water.
- (ii) Latent heat of steam.
- (iii) Dryness fraction of steam.
- (iv) Enthalpy of wet steam.

2. Attempt any **TWO** parts of the following : (10×2=20)

(a) In a constant volume 'Otto Cycle' the pressure at the end of compression is 15 times that at the start, the temperature of air at the beginning of compression is 38°C and maximum temperature attained in cycle is 1950°C. Determine :

- (i) Compression ratio
- (ii) Thermal Efficiency of the Cycle
- (iii) Work done.

Take γ for air = 1.4.

(b) (i) Describe the different operations of Rankine cycle. Also derive the expression for its efficiency.

(ii) State the methods of increasing the thermal efficiency of a Rankine cycle.

(c) Explain the working of a two-stroke and four stroke S.I. Engines with the help of neat sketch.

3. Attempt any **TWO** parts of the following : (10×2=20)

(a) A roller of radius 200 mm and weight 1732 N is to be pulled over a curb of height 100 mm by a horizontal force P applied to the end of string wound tightly around the circumference of the roller. Find the

magnitude of P required to start the roller move over the curb. Also find the least pull through the centre O of the wheel to just turn the roller over the curb.

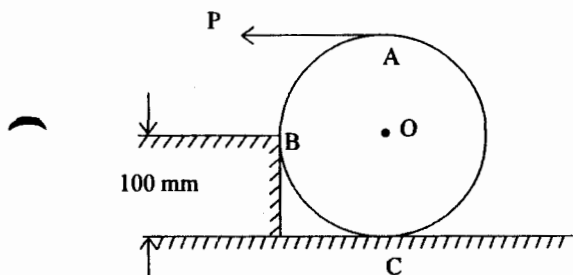


Fig. 1

- (b) A ladder of length L rests against a wall, the angle of inclination being 45° . If coefficient of friction between the ladder and ground and that being ladder and wall be 0.5 each. What will be maximum distance on the ladder to which a man whose weight is 1.5 times the weight of the ladder may ascend before the ladder begins to slip.

- (c) Derive the relation $\frac{T_1}{T_2} = e^{\mu\theta}$ for a flat belt drive.

4. Attempt any **TWO** parts of the following : (10×2=20)

- (a) (i) Discuss the various types of beams and loading.
 (ii) Derive the relation between load intensity, shear force and bending moment.

- (b) Draw the SFD and BMD for the beam shown in Fig. 2:

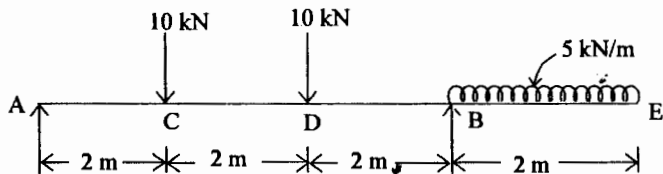


Fig. 2

Also locate the point of contraflexure.

- (c) Find the forces in each member of the cantilever truss as shown in Fig. 3.

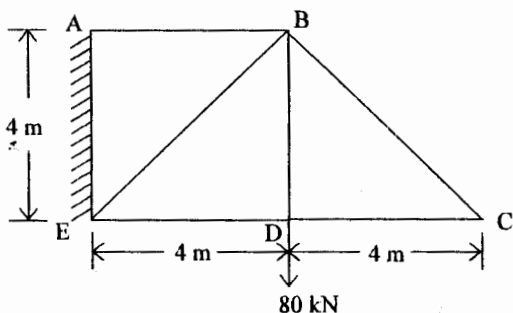


Fig. 3

5. Attempt any **FOUR** parts : (5×4=20)
- Derive the relationship between Young's modulus and modulus of rigidity.
 - A steel bar 4 cm is section 3 m long is subjected to an axial pull of 128 kN. Taking $E = 200 \text{ GN/m}^2$, calculate alteration in the length of the bar. Calculate also the amount of energy stored in the bar during the extension.
 - The principal stresses at a point across two perpendicular planes are 75 MN/m^2 (tensile) and 35 MN/m^2 (Tensile). Find normal, tangential stresses and resultant stress and its obliquity on a plane at 20° with the major principal plane.
 - Derive the bending equation :

$$\frac{M}{I} = \frac{\sigma}{y} = \frac{E}{R}$$

Also state the assumptions.

- A hollow shaft of diameter ratio $3/8$ is required transmit 600 kW at 110 rpm, maximum torque being 20% greater than the mean. The shear stress is not to exceed 63 MN/m^2 and twist in a length of 3 m not to exceed 1.4° . Calculate the maximum external diameter satisfying these conditions.